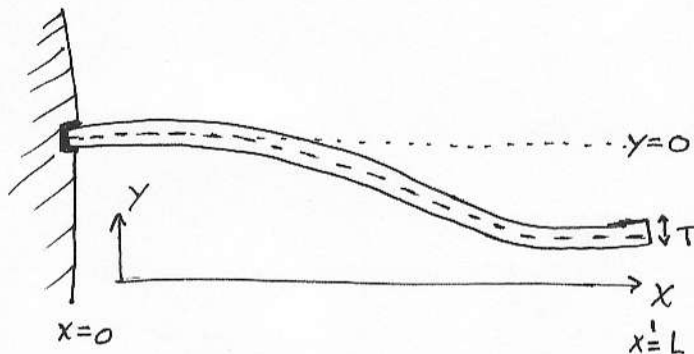


Q. What is the shape and frequency of a stiff sheet undergoing small-amplitude oscillations about a flat equilibrium geometry? Ignore gravity.

R. Jones

\* Assumptions: the sheet is held fixed, with fixed slope, at one end, taken to be the left end. The right end is free, as shown below.

\* Coordinate system:  $x, y$  axes, with origin, shown in the figure to the right.



\* Dimensions: the plank is of length  $L$ , thickness  $T$ , width  $W$ , where  $T$  and  $W$  functions of  $x$  and not necessarily constants. The plank is made of a uniform material of density  $\rho$  and bulk modulus  $E$ .

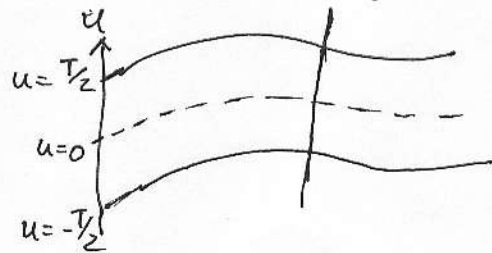
\* Analysis of static forces at fixed displacement

Make an imaginary cut through the plank at  $x$  and ask what forces the part at the left places on that to the right

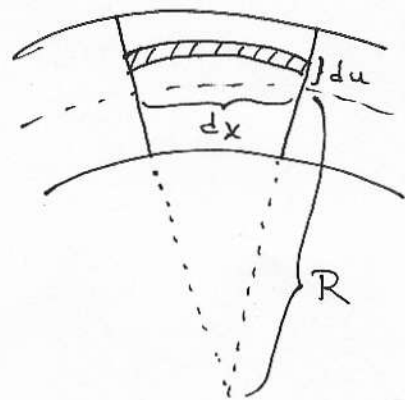
Let  $u$  measure depth inside the plank relative to the central layer.

Forces are tension ( $F_x$ ) and shear

( $F_y$ ).  $F_x$  arises from bending of the material and its bulk elasticity.  $F_y$  arises from the fact that the plank remains a single continuous piece - it is a constraint in the problem.



To get  $F_x$ , consider a bent shape as shown to the right. The layer indicated by  $du$  is stretched from its equilibrium length  $dx$  by a distance



$$\Delta x = \left(\frac{R+u}{R}\right)dx - dx = \frac{u}{R}dx$$

What is  $R$ ? It is given by the second derivative for  $y(x)$

Let  $y - y_0 = \frac{1}{2} y'' dx^2$ , then  $R^2 = dx^2 + (y_0 + \frac{1}{2} y'' dx^2)^2$

But  $y_0 = R$ , so  $R^2 = dx^2 + R^2 + R y'' dx^2 + O(dx^2)^2$

$$\therefore y'' = -\frac{1}{R}, \quad R = -\frac{1}{y''}$$

So  $\Delta x = -u y'' dx$ . What is the force element from  $du$ ?

$$dF_x = E \frac{\Delta x}{dx} du W = -WE u y'' du$$

$$F_x = \int_{-T/2}^{T/2} \frac{dF_x}{du} du = -WE y'' \left(\frac{u^2}{2}\right) \Big|_{-T/2}^{T/2} = -\frac{1}{8} WE y'' (T^2 - T^2) = 0$$

Forces cancel, but torques do not. Tension forces make torque

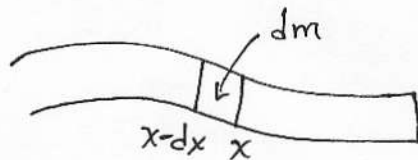
$$\Gamma = \int_{-T/2}^{T/2} \frac{dF_x}{du} u du = -WE y'' \left(\frac{u^3}{3}\right) \Big|_{-T/2}^{T/2} = -\frac{1}{12} WE y'' T^3$$

To propagate this force along the plank, subject to the constraint that the plank remains intact, we need to keep track of both the shear  $S(x)$  and torque  $\Gamma(x)$  as we move along.

$$S(x-dx) = S(x) + \ddot{y}(x) dm$$

$$\Gamma(x-dx) = \Gamma(x) + S(x) dx$$

$$\left. \begin{aligned} S' &= -\rho W T \ddot{y} \\ \Gamma' &= -S \end{aligned} \right\} \Gamma'' = \rho W T \ddot{y}$$



$$dm = \rho W T dx$$

assume "thin plank"

$$L \gg T$$

$$E \frac{d^2}{dx^2} \left( WT^3 \frac{d^2 y}{dx^2} \right) = -\rho WT \frac{d^2 y}{dt^2}$$

This is not a wave equation because it is fourth order in  $x$ , but it can support wave-like solutions under certain conditions.

Case a) Let  $W, T$  be constants, then  $W$  drops out!

$$\frac{d^4 y}{dx^4} = -\left(\frac{\rho}{ET^2}\right) \frac{d^2 y}{dt^2}$$

Let  $y = y_0 e^{i(kx - \omega t)}$ ,  $k^4 y_0 = \frac{\rho \omega^2}{ET^2} y_0$

Boundary conditions are  $y = y' = 0$  @  $x = 0$   
 $y'' = y''' = 0$  @  $x = L$

Normal mode condition:  $y(x, t) = y(x) e^{i\omega t}$

$$\frac{d^4 y}{dx^4} = +\left(\frac{\rho}{ET^2}\right) \omega^2 y$$

$$y = y_1 e^{ikx} + y_2 e^{-ikx} + y_3 e^{kx} + y_4 e^{-kx}, \quad k = \left(\frac{\rho \omega^2}{ET^2}\right)^{1/4}$$

①  $y_1 + y_2 + y_3 + y_4 = 0$

②  $i(y_1 - y_2) + (y_3 - y_4) = 0$

③  $-iy_1 e^{ikL} + y_2 e^{-ikL} + y_3 e^{kL} + y_4 e^{-kL} = 0$

④  $-iy_1 e^{ikL} + iy_2 e^{-ikL} + y_3 e^{kL} - y_4 e^{-kL} = 0$

These equations are homogeneous, so solutions only exist subject to certain constraints on  $k$  (or  $\omega$ ). This is what gives the eigenmode equation.

$$\begin{vmatrix} 1 & 1 & 1 & 1 \\ i & -i & 1 & -1 \\ -e^{ikL} & -e^{-ikL} & e^{kL} & e^{-kL} \\ -ie^{ikL} & ie^{-ikL} & e^{kL} & -e^{-kL} \end{vmatrix} = 0$$

Let  $a = e^{ikL}$ ,  $b = e^{kL}$

$$\begin{aligned} &= \left\{ -i[-1-1] - 1\left[\frac{1}{ab} - \frac{i}{ab}\right] - 1\left[-\frac{b}{a} - i\frac{b}{a}\right] \right\} \\ &- \left\{ i[-1-1] - 1\left[\frac{a}{b} + i\frac{a}{b}\right] - 1[-ab + iab] \right\} \\ &+ \left\{ i\left[\frac{1}{ab} - \frac{i}{ab}\right] + i\left[\frac{a}{b} + i\frac{a}{b}\right] - 1[-i - i] \right\} \\ &- \left\{ i\left[-\frac{b}{a} - i\frac{b}{a}\right] + i[-ab + iab] + 1[-i - i] \right\} \end{aligned}$$

$$= 8i + \frac{2i}{ab} + \frac{2ib}{a} + \frac{2ia}{b} + 2iab$$

$$\therefore 4ab + 1 + b^2 + a^2 + a^2b^2 = 0$$

$$(a+b)^2 + (ab+1)^2 = 0$$

$\therefore a=1, b=-1$  or  $a=-1, b=1$ , neither one is possible!  
or  $a$  is complex! in fact, it is a pure phase  $e^{i\phi_a}$

$$(ab+1) = \pm i(a+b), \quad b \text{ is pure real, } b > 1.$$

$$\left. \begin{array}{l} \textcircled{1} a_r + b = \pm a_i b \\ \textcircled{2} a_r b + 1 = \mp a_i \end{array} \right\} \frac{a_r}{b} + a_r b + 2 = 0 \Rightarrow a_r = \frac{-2b}{b^2+1}$$

$$\phi_a = \tan^{-1}\left(\frac{a_i}{a_r}\right) = \tan^{-1}\left(\frac{b^2-1}{2b}\right) = \pm \ln b \quad a_i = \pm \frac{1-b^2}{1+b^2}$$